

Vol's Chiasson

69  
Navaz

Pourmovahed

Name B. / ... 7. 11

INSTRUCTOR (Check One Above)

MECH-422 ENERGY SYSTEMS LABORATORY

Midterm Exam

November 3, 2004

83

Instructions:

1. Write your name at the top of **each** page and check your Lab Instructor's name.
2. This exam is open-book, open notes/manual. However, **no sharing of material during the exam is allowed among students.**
3. You have 2 hours. This exam consists of 10 multiple-choice problems/questions (5 points each) and 2 work-out problems (25 points each).
4. Should questions arise, remain at your desk and raise your hand.
5. **For multiple-choice problems**, if differences in round-off procedures result in slight numerical differences between the answers given and your results, **circle the closest answer.**

Barometric pressure = 14.7 psia = 101 kPa for all problems (unless specified otherwise)

$g = 9.81 \text{ m/s}^2 = 32.2 \text{ ft/s}^2$   $\text{lb}_f = 32.2 \text{ lb}_m\text{-ft/s}^2$  specific heat ratio for air,  $k=1.4$

ideal gas law:  $P = \rho RT$ ,  $R = 0.287 \text{ kJ}/(\text{kg K})$  for air

density of water,  $\rho = 1000 \text{ kg/m}^3 = 1.94 \text{ slugs/ft}^3 = 62.4 \text{ lb}_m/\text{ft}^3$

density of air =  $1.23 \text{ kg/m}^3 = 0.00239 \text{ slugs/ft}^3 = 0.075 \text{ lb}_m/\text{ft}^3$

density of mercury =  $13,550 \text{ kg/m}^3$

dynamic viscosity of air =  $1.79 \times 10^{-5} \text{ N s/m}^2 = 3.82 \times 10^{-7} \text{ lb}_f \text{ s / ft}^2$

dynamic viscosity of water =  $1.79 \times 10^{-3} \text{ N s/m}^2 = 2.037 \times 10^{-5} \text{ lb}_f \text{ s / ft}^2$

1 Btu/s = 1.055 kW      550 ft lb<sub>f</sub>/s = 1 hp

1 kg = 0.068522 slugs    1 gallon = 0.1337 ft<sup>3</sup>    1 liter =  $3.53 \times 10^{-2} \text{ ft}^3 = 10^{-3} \text{ m}^3$

universal gas constant,  $R_u = 8.314 \text{ kJ/Kmol.K}$

molecular mass of hydrogen: 2.016

molecular mass of oxygen: 31.999

**Problem 1 (5 points)**

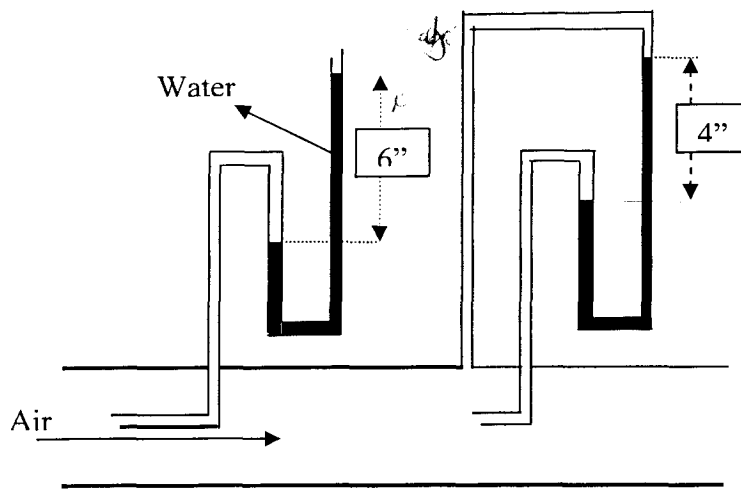
The volume of a cone in terms of its radius at the base and its height is given by  $V = \frac{1}{3} \pi R^2 h$ . The radius was measured to be 2 in  $\pm$  0.01 and the height was measured to be 8 in  $\pm$  0.01. The most pessimistic relative and absolute errors in the measurement of volume are:

- (a) 0.88%      33.510 in<sup>3</sup>
- (b) 1.13%      0.377 in<sup>3</sup>
- (c) 0.01%      3.351 in<sup>3</sup>
- (d) 11.25%     3.770 in<sup>3</sup>

*SV*  
 $\frac{\Delta V}{V} = 2 \frac{\Delta R}{R} + \frac{\Delta h}{h}$   
 $= 2 \frac{0.01}{2} + \frac{0.01}{8} = 0.01125 = 1.13\%$   
 $\Delta V = 1.13\% \times 33.510 = 0.377 \text{ in}^3$

**Problem 2 (5 points)**

With reference to the following figure, the stagnation, static, and dynamic pressures are...



- a. 6", 4" and 6" of water.
- b. 10", 4" and 2" of water.
- c. 4", 10" and 6" of water.
- (d) 6", 2" and 4" of water.

✓ Chiasson

Navaz

Pourmovahed

Name B. Navaz

**Problem 3 (5 points)**

In the pipe flow experiment, the pressure drop between two points that are 5 ft apart is measured to be 2" of water. If the diameter of the pipe is 1.75" and the average velocity is calculated to be 80 ft/sec, the friction factor for the pipe is about.....

- (a) 0.04
- (b) 0.48
- (c) 0.20
- (d) 0.02

$$h_f = f \frac{L}{D} \frac{V^2}{2g}$$

$$2 \text{ in} = f \frac{5 \text{ ft}}{1.75 \text{ in}} \frac{(80 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2}$$

$$h_f = f \frac{L}{D} \frac{V^2}{2g}$$

$$2 \text{ in} = f \frac{5 \text{ ft}}{1.75 \text{ in}} \frac{(80 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2}$$

$$f = \frac{2 \text{ in} \times 1.75 \text{ in} \times 2 \times 32.2 \text{ ft/sec}^2}{5 \text{ ft} \times (80 \text{ ft/sec})^2}$$

$$f = 0.04$$

**Problem 4 (5 points)**

In the Pipe Flow experiment, the average velocity of air is calculated to be 80 ft/sec. The static pressure after the fan is measured to be 4.5" of water. Taking  $\alpha = 1.0$ , what is the fan's head?

- (a) 304 ft
- (b) 100 ft
- (c) 404 ft
- (d) 204 ft

$$h_p = \alpha \frac{V^2}{2g}$$

$$h_p = 1.0 \frac{(80 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2}$$

$$h_p = 99.07 \text{ ft}$$

$$h_p = \alpha \frac{V^2}{2g}$$

$$h_p = 1.0 \frac{(80 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2}$$

$$h_p = 99.07 \text{ ft}$$

✓ Chiasson

Navaz

Pourmovahed

Name Chiasson, Navaz Pourmovahed

**Problem 5 (5 points)**

For the **Flow Meters** experiment, the following data is available:

- ✗ Pipe diameter = 3"
- Laminar flow meter reading: 2.6" of water

If the laboratory is at 70 °F, what is the **actual mass flow rate** of the air as determined by the laminar flow meter?  $\rho_{\text{air @ 70 °F and 530 °R}} = 0.076314 \text{ lbm/ft}^3$

- (a) 35.000 lb<sub>m</sub>/min
- (b) 0.583 lb<sub>m</sub>/min
- (c) 2.671 lb<sub>m</sub>/min
- (d) 6.977 lb<sub>m</sub>/min

*Handwritten notes for Problem 5:*

$m = \rho V A$

$V = \frac{h}{12} \times \frac{\pi}{4} \times \left(\frac{3}{12}\right)^2$

$V = \frac{2.6}{12} \times \frac{\pi}{4} \times \frac{9}{144}$

$V = 0.00377 \text{ ft}^3$

$m = 0.076314 \times 0.00377$

$m = 0.000285 \text{ lbm}$

$\frac{0.000285}{60} = 4.75 \times 10^{-6} \text{ lbm/min}$

Δ **Problem 6 (5 points)**

In the **Flow Meters** experiment, the following data is available:

- Pipe diameter = 3"
- Venturi throat diameter = 1.2"
- ΔP = 1.5" of water for the Venturi

What is the **ideal** volumetric flow rate?

- (a) 94.5 ft<sup>3</sup>/min
- (b) 67.0 ft<sup>3</sup>/min
- (c) 1.6 ft<sup>3</sup>/min
- (d) 79 ft<sup>3</sup>/min

*Handwritten notes for Problem 6:*

$A_1 = \left(\frac{3}{12}\right)^2 \times \frac{\pi}{4}$

$A_2 = \left(\frac{1.2}{12}\right)^2 \times \frac{\pi}{4}$

$\frac{A_1}{A_2} = \left(\frac{3}{1.2}\right)^2 = 6.25$

$V_1 = 6.25 V_2$

$\Delta P = 1.5 \text{ in H}_2\text{O}$

\_\_\_\_ Chiasson

\_\_\_\_ Navaz

\_\_\_\_ Pourmovahed

Name \_\_\_\_\_

**Problem 7 (5 points)**

Which of the following statements is correct?

- ~~(a)~~ Venturi meter is the best flow meter because it gives the minimum pressure drop.
- (b) Orifice plate yields the most accurate volumetric flow rate because of the sudden change of area.
- ~~(c)~~ The elbow is the least effective flow measurement device because it gives the minimum pressure drop.
- (d) The elbow is the least effective flow measurement device because the flow turns by  $90^\circ$ .

**Problem 8 (5 points)**

In the **Centrifugal Pump** experiment, the following data are taken:

Pump head = 4.0 m

$Q = 0.3$  lit/sec

Speed = 2000 rpm

Torque = 0.2 Nm

The efficiency of the pump is about.....

- (a) 0.46%                      (b) 28%                      (c) 4.6%                      (d) 46%

✓ Chiasson

Navaz

Pourmovahed

Name

B. J. T. H.

**Problem 9 (5 points)**

The specific speed of a pump represents:

- Speed required to produce unit head at unit volumetric flow rate
- A combination of  $C_Q$  and  $C_H$  coefficients to obtain the speed of the pump
- The speed of the pump per unit mass of the flow rate
- The speed of the pump to produce a required head

**Problem 10 (5 points)**

In the **Centrifugal Pump** experiment the following data is available:

Speed = 2000 rpm

Torque = 0.2 Nm

Impeller diameter = 5.5 cm

The power coefficient is about....

(a) 6.605

(b) 0.005

(c) 0.02

(d) 0.009

Name Navaz Pourmovahed

**Problem 12 (25 points)**

The data below were obtained in the test cell for the **actual** engine used in the **Road Load Simulation** experiment:

road speed mph	road speed km/h	engine power kW	engine speed rpm	engine torque N.m	fuel flow rate kg/hr	air flow rate kg/hr	exhaust temp. C	coolant inlet temp. C	coolant outlet temp. C	coolant flow rate L/min
4 <sup>th</sup> gear										
70	112.7	25.8	2113.0	116.6	7.1	102.0	602.2	7.5	65.0	5.3

Determine:

- (a) The actual thermal efficiency of this engine (%).

Handwritten calculation for (a):  

$$\eta_{th} = \frac{P_{out}}{\dot{m}_f \cdot \text{LHV}} = \frac{25.8 \text{ kW}}{7.1 \text{ kg/hr} \cdot 42.5 \text{ MJ/kg}} = 71.1\%$$
 (Note: 42.5 MJ/kg is handwritten, likely representing a value for LHV)

- (b) Energy carried away by the exhaust gases as a percentage of the heat released by the fuel (%).

Handwritten calculation for (b):  

$$\dot{m}_{exh} c_p (T_{exh} - T_{amb}) = (10.5 \text{ kg/hr}) (1.0) (602.2 - 25) = 6240 \text{ kJ/hr}$$

$$\frac{6240}{55741.6} = 11.2\%$$
 (Note: 10.5 kg/hr is handwritten, likely representing the sum of fuel and air flow rates)

- (c) Heat carried away by the coolant as a percentage of the heat released by the fuel (%).

Handwritten calculation for (c):  

$$\dot{m}_c c_p (T_{out} - T_{in}) = 5.3 \text{ L/min} \cdot 1 \text{ kg/L} \cdot (65 - 7.5) = 2034 \text{ kJ/min}$$

$$= 2034 \cdot 60 = 122040 \text{ kJ/hr}$$

$$\frac{122040}{55741.6} = 21.9\%$$
 (Note: 55741.6 is handwritten, likely representing the total heat release rate)

25

✓ Chiasson

Navaz

Pourmovahed

Name Ray. T.

**Problem 11 (25 points)**

The following data were collected for the PEM Fuel Cell Systems actually used in the Lab:

H <sub>2</sub> Flow Rate (ml/min)	150	Air Flow Rate (ml/min)	1000			
H <sub>2</sub> Supply Pressure (psig)	20	Air Supply Pressure (psig)	20			
Stack Temp. (°C)	50	Hydrogen Humidifier Temp. (°C)	30			
Air Humidifier Temp. (°C)	20	Total Electrode Area (cm <sup>2</sup> )	16			
Current (Amps)	Cell Potential (V)			Temp. (°C)		
	Cell 1	Cell 2	Cell 3	Cell 1	Cell 2	Cell 3
10	0.4	0.5	0.0	53.8	56.0	55.3

18.75

a. What is the actual **stack** efficiency (%).

Handwritten calculations for part a:

$\eta = \frac{VI}{\dot{m}_H \cdot \dot{m}_O} = \frac{10 \cdot 0.4}{150 \cdot 1000} = 0.00267$

$\eta = \frac{10 \cdot 0.5}{150 \cdot 1000} = 0.00333$

$\eta = \frac{10 \cdot 0.0}{150 \cdot 1000} = 0$

Final result:  $\eta = 0.00267 \cdot 100 = 0.267\%$

~~7.55% (crossed out)~~

b. Estimate the theoretical open-cell voltage.

Handwritten calculations for part b:

$E = \frac{1.23}{1.05} = 1.171 \text{ V}$

$E = 1.23 - \frac{0.059}{2} \log \left( \frac{1}{1000} \right) = 1.23 - 0.0885 = 1.1415 \text{ V}$

Final result:  $E = 1.14 \text{ V}$