

Name Solution

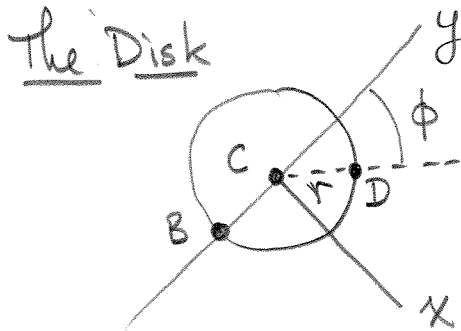
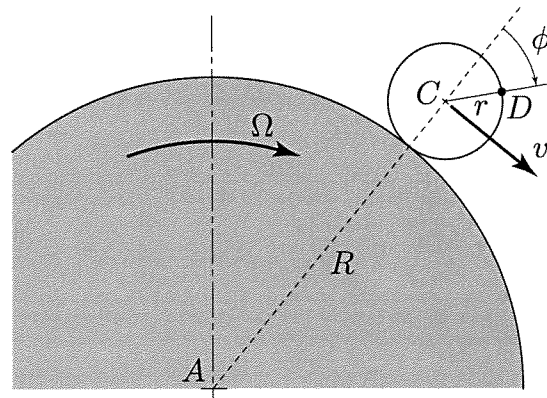
THE PENNSYLVANIA STATE UNIVERSITY
Department of Engineering Science and Mechanics

Engineering Mechanics 112H—Exam 3
April 24, 2006

Problem 1	Problem 2	Problem 3	Problem 4	Total

Problem 1 (31 pts)

The disk of radius r rolls without slipping over the exterior of a large drum of radius R . The drum rotates clockwise at constant angular speed Ω . In the position shown, the center of the disk has speed v , which is increasing at the rate \dot{v} . Derive expressions for the velocity and acceleration of point D , which is situated at an arbitrary angle ϕ relative to the radial line connecting the centers.



We are given that: $\vec{v}_C = v \hat{i}$

In addition, since C is moving in a circle centered at A

$$\vec{a}_C = \dot{v} \hat{i} - \frac{v^2}{R+r} \hat{j}$$

Since C rolls without slip on the drum, we also know

$$\vec{v}_B = R\Omega \hat{i}. \text{ Now:}$$

$$\vec{v}_C = \vec{v}_B + \vec{\omega}_C \times \vec{r}_{C/B} \Rightarrow v \hat{i} = R\Omega \hat{i} + \omega_C \hat{k} \times r \hat{j}$$

$$\Rightarrow v = R\Omega - r\omega_C \Rightarrow \omega_C = \frac{R\Omega - v}{r}$$

Since ω_C is valid for all time:

$$1 \quad \alpha_C = \dot{\omega}_C = \frac{1}{r}(R\dot{\Omega} - \dot{v})$$

$$\text{since } \Omega = \text{const} \Rightarrow \alpha_c = -\frac{\dot{v}}{r}$$

$$\text{Now: } \vec{V}_D = \vec{V}_c + \vec{\omega}_c \times \vec{r}_{D/c}$$

$$\begin{aligned} \vec{V}_D &= v \hat{i} + \frac{1}{r}(R\Omega - v) \hat{k} \times r(\sin\phi \hat{i} + \cos\phi \hat{j}) \\ &= [v - (R\Omega - v)\cos\phi] \hat{i} + (R\Omega - v)\sin\phi \hat{j} \end{aligned}$$

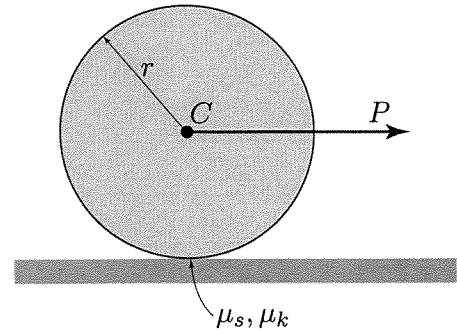
$$\boxed{\vec{V}_D = [v(1 + \cos\phi) - R\Omega \cos\phi] \hat{i} + (R\Omega - v)\sin\phi \hat{j}}$$

$$\begin{aligned} \vec{a}_D &= \vec{a}_c + \vec{\alpha}_c \times \vec{r}_{D/c} - \omega_c^2 \vec{r}_{D/c} \\ &= \dot{v} \hat{i} - \frac{v^2}{R+r} \hat{j} - \frac{\dot{v}}{r} \hat{k} \times r(\sin\phi \hat{i} + \cos\phi \hat{j}) \\ &\quad - \left(\frac{R\Omega - v}{r}\right)^2 r(\sin\phi \hat{i} + \cos\phi \hat{j}) \end{aligned}$$

$$\boxed{\begin{aligned} \vec{a}_D &= \left[\dot{v} + \dot{v} \cos\phi - \frac{(R\Omega - v)^2}{r} \sin\phi \right] \hat{i} \\ &\quad + \left[-\frac{v^2}{R+r} - \dot{v} \sin\phi - \frac{(R\Omega - v)^2}{r} \cos\phi \right] \hat{j} \end{aligned}}$$

Problem 2 (31 pts)

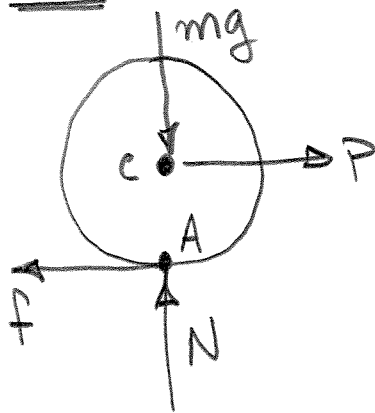
The uniform disk of radius r and mass m is at rest when a force P begins pulling to the right on its center. Given that the coefficients of static and kinetic friction between the disk and the ground are μ_s and μ_k , respectively, and that P is sufficiently large to cause the disk to slip,



- determine the angular acceleration of the disk and the acceleration of its mass center C ,
- state the condition for the admissibility of this solution, and
- determine the condition on P that satisfies the admissibility condition stated in (b).

Newton-Euler

FBD



$$\Sigma F_x: P - f = ma_{cx}$$

$$\Sigma F_y: N - mg = ma_{cy}$$

$$\Sigma M_c: -fr = I_c \alpha$$

Material: $f = \mu_k N$; $I_c = \frac{1}{2}mr^2$

Kinematics: $a_{cy} = 0$

Solving: $N = mg$

$$P - \mu_k mg = ma_{cx} \Rightarrow a_{cx} = \frac{P}{m} - \mu_k g$$

$$-\mu_k mgr = \frac{1}{2}mr^2 \alpha \Rightarrow \alpha = -2 \frac{\mu_k g}{r}$$

The admissibility condition concerns the motion at A. Since f is to the left, we must have that $a_{Ax} \geq 0$:

$$\vec{a}_A = \vec{a}_c + \vec{\alpha} \times \vec{r}_{A/c} \quad (\omega = 0)$$

$$a_{Ax} \hat{i} + a_{Ay} \hat{j} = \left(\frac{P}{m} - \mu_k g\right) \hat{i} - 2 \frac{\mu_k g}{r} \hat{k} \times (-r \hat{j})$$

$$\hat{i}: a_{Ax} = \frac{P}{m} - \mu_k g - 2\mu_k g = \frac{P}{m} - 3\mu_k g$$

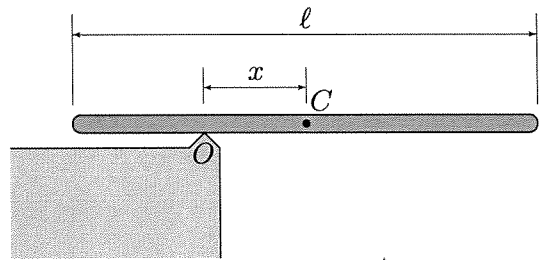
Since we want $a_{Ax} > 0$

$$\Rightarrow \boxed{P \geq 3\mu_k mg}$$

It is also true that $a_{cx} \geq 0$, but it is easily seen that this is also satisfied by the above condition.

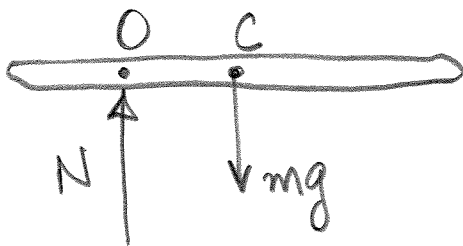
Problem 3 (31 pts)

The uniform slender bar is released from rest in the horizontal position shown. Determine the value of x for which the angular acceleration is a maximum, and determine the corresponding angular acceleration α .



Need to find $\alpha(x)$ and then take $\frac{d\alpha}{dx} = 0$.

FBD \curvearrowright_x



$$\sum M_O: -mgx = I_O \alpha$$

Material: $I_O = I_C + mx^2$

$$I_O = \frac{1}{12}ml^2 + mx^2$$

$$= m(x^2 + \frac{l^2}{12})$$

Therefore: $\alpha = -g \frac{x}{x^2 + \frac{l^2}{12}}$

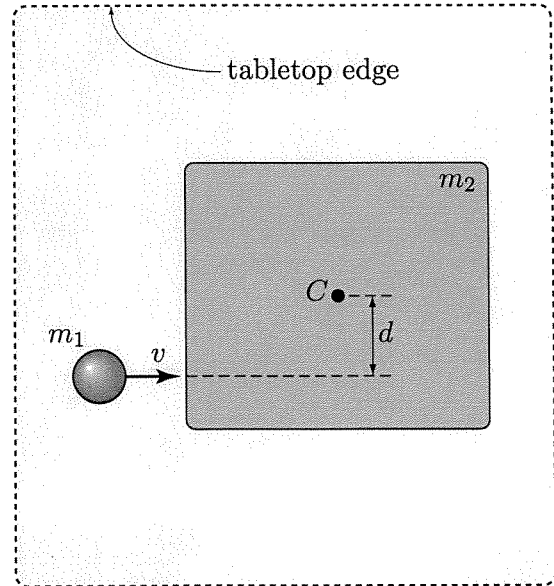
$$\frac{d\alpha}{dx} = -g \left[\frac{x^2 + \frac{l^2}{12} - 2x^2}{(x^2 + \frac{l^2}{12})^2} \right] = 0$$

$$-x^2 = -\frac{l^2}{12} \Rightarrow \boxed{x = \frac{l}{\sqrt{12}} = \frac{l}{2\sqrt{3}} = 0.289l}$$

$$\boxed{\alpha_{\max} = -g \frac{\frac{l}{\sqrt{12}}}{\frac{l^2}{12} + \frac{l^2}{12}} = -\sqrt{3} \frac{g}{l}}$$

Problem 4 (7 pts)

The box of mass m_2 is at rest on a smooth horizontal tabletop when it is hit by a non-spinning ball of mass m_1 traveling at speed v , which then bounces off. The mass center of the box is at C and its mass is uniformly distributed. Describe the ensuing motion of the box. Be brief, but careful.



The impulsive force on the box due to m_1 is to the right and has no vertical component. In addition, its line of action does not go through C . Therefore, the box will rotate ccw and C will move to the right.