

Problem 14-17

Gases from an incinerator are released to the atmosphere using a stack that is 0.6 m in diameter and 10 m high. The outer surface of the stack is at 40°C and the surrounding air is at 10°C. Determine the rate of heat transfer from the stack assuming a) no wind, and b) a wind at 20 km/hr.

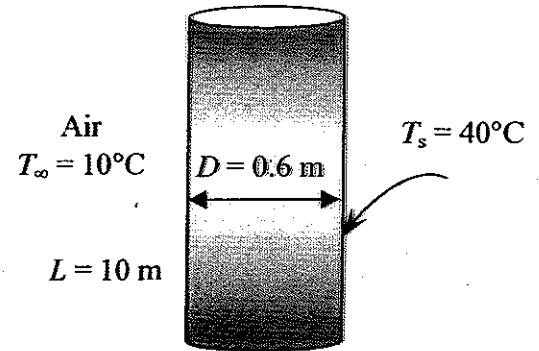
**Properties** The properties of air at 1 atm and the film temperature of  $(T_s + T_\infty)/2 = (40 + 10)/2 = 25^\circ\text{C}$  are (Table A-22)

$$k = 0.02551 \text{ W/m}\cdot^\circ\text{C}$$

$$\nu = 1.562 \times 10^{-5} \text{ m}^2/\text{s}$$

$$\text{Pr} = 0.7296$$

$$\beta = \frac{1}{T_f} = \frac{1}{(25 + 273)\text{K}} = 0.003356 \text{ K}^{-1}$$



**Analysis (a)** When there is no wind heat transfer is by natural convection. The characteristic length in this case is the height of the stack,  $L_c = L = 10 \text{ m}$ . Then,

$$\begin{aligned} Ra &= \frac{g\beta(T_s - T_\infty)L^3}{\nu^2} \text{Pr} \\ &= \frac{(9.81 \text{ m/s}^2)(0.003356 \text{ K}^{-1})(40 - 10 \text{ K})(10 \text{ m})^3}{(1.562 \times 10^{-5} \text{ m}^2/\text{s})^2} (0.7296) = 2.953 \times 10^{12} \end{aligned}$$

We can treat this vertical cylinder as a vertical plate since

$$\frac{35L}{Gr^{1/4}} = \frac{35(10)}{(2.953 \times 10^{12} / 0.7296)^{1/4}} = 0.246 < 0.6 \quad \text{and thus } D \geq \frac{35L}{Gr^{1/4}}$$

The Nusselt number is determined from

$$Nu = 0.1Ra^{1/3} = 0.1(2.953 \times 10^{12})^{1/3} = 1435$$

Then

$$h = \frac{k}{L_c} Nu = \frac{0.02551 \text{ W/m}\cdot^\circ\text{C}}{10 \text{ m}} (1435) = 3.660 \text{ W/m}^2\cdot^\circ\text{C}$$

and

$$\dot{Q} = hA(T_s - T_\infty) = (3.660 \text{ W/m}^2\cdot^\circ\text{C})(\pi \times 0.6 \times 10 \text{ m}^2)(40 - 10)^\circ\text{C} = 2070 \text{ W}$$

Problem 14-17 (Continued)

Gases from an incinerator are released to the atmosphere using a stack that is 0.6 m in diameter and 10 m high. The outer surface of the stack is at 40°C and the surrounding air is at 10°C. Determine the rate of heat transfer from the stack assuming a) no wind, and b) a wind at 20 km/hr.

(b) When the stack is exposed to 20 km/h winds

$$\text{Re} = \frac{VD}{\nu} = \frac{(20 \times 1000 / 3600 \text{ m/s})(0.6 \text{ m})}{1.562 \times 10^{-5} \text{ m}^2/\text{s}} = 213,400$$

$$\text{Nu} = 0.027 \text{Re}^{0.805} \text{Pr}^{1/3} = 0.027(213,400)^{0.805} (0.7296)^{1/3} = 473.9$$

$$h = \frac{k}{D} \text{Nu} = \frac{0.02551 \text{ W/m}\cdot\text{°C}}{0.6 \text{ m}} (473.9) = 20.15 \text{ W/m}^2\cdot\text{°C}$$

$$\dot{Q} = hA(T_s - T_\infty) = (20.15 \text{ W/m}^2\cdot\text{°C})(\pi \times 0.6 \times 10 \text{ m}^2)(40 - 10)\text{°C} = 11,390 \text{ W}$$

There is more than five-fold increase in heat transfer due to winds.

### Problem 14-19

A 10-m-long section of a 6-cm-diameter horizontal hot-water pipe passes through a large room whose temperature is 27°C. If the temperature and the emissivity of the pipe are 73°C and 0.8, determine the rate of heat loss from the pipe by a) natural convection and b) radiation.

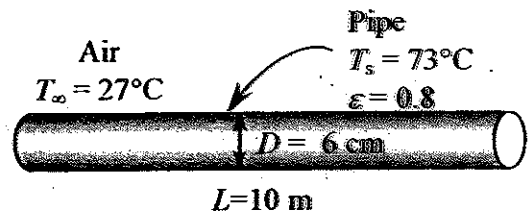
**Properties** The properties of air at 1 atm and the film temperature of  $(T_s + T_\infty)/2 = (73 + 27)/2 = 50^\circ\text{C}$  are (Table A-22)

$$k = 0.02735 \text{ W/m}\cdot^\circ\text{C}$$

$$\nu = 1.798 \times 10^{-5} \text{ m}^2/\text{s}$$

$$\text{Pr} = 0.7228$$

$$\beta = \frac{1}{T_f} = \frac{1}{(50 + 273)\text{K}} = 0.003096 \text{ K}^{-1}$$



The characteristic length  $L_c = D = 0.06 \text{ m}$ .

$$\text{Ra} = \frac{g\beta(T_s - T_\infty)D^3}{\nu^2} \text{Pr}$$

$$= \frac{(9.81 \text{ m/s}^2)(0.003096 \text{ K}^{-1})(73 - 27 \text{ K})(0.06 \text{ m})^3}{(1.798 \times 10^{-5} \text{ m}^2/\text{s})^2} (0.7228) = 6.747 \times 10^5$$

(From Eq. 14-25, Page 619)

$$\text{Nu} = \left\{ 0.6 + \frac{0.387 \text{Ra}^{1/6}}{\left[ 1 + (0.559/\text{Pr})^{9/16} \right]^{8/27}} \right\}^2 = \left\{ 0.6 + \frac{0.387(6.747 \times 10^5)^{1/6}}{\left[ 1 + (0.559/0.7228)^{9/16} \right]^{8/27}} \right\}^2 = 13.05$$

$$h = \frac{k}{D} \text{Nu} = \frac{0.02735 \text{ W/m}\cdot^\circ\text{C}}{0.06 \text{ m}} (13.05) = 5.950 \text{ W/m}^2\cdot^\circ\text{C}$$

$$A_s = \pi DL = \pi(0.06 \text{ m})(10 \text{ m}) = 1.885 \text{ m}^2$$

$$\dot{Q} = hA_s(T_s - T_\infty) = (5.950 \text{ W/m}^2\cdot^\circ\text{C})(1.885 \text{ m}^2)(73 - 27)^\circ\text{C} = 516 \text{ W}$$

(b) The radiation heat loss from the pipe is

$$\begin{aligned} \dot{Q}_{\text{rad}} &= \varepsilon A_s \sigma (T_s^4 - T_{\text{surr}}^4) \\ &= (0.8)(1.885 \text{ m}^2)(5.67 \times 10^{-8} \text{ W/m}^2\cdot\text{K}^4) \left[ (73 + 273 \text{ K})^4 - (27 + 273 \text{ K})^4 \right] = 533 \text{ W} \end{aligned}$$