

1. The top surface of a hot plate 8 m long and 2.5 m wide with a surface temperature of 120°C is to be cooled by forced air at 6 m/sec and 20°C. Assuming an atmospheric pressure of 1 atm, a critical Reynold's number of  $5 \times 10^5$ , and steady convective heat transfer, determine a) the convective heat transfer coefficient  $h$ , and b) the heat transfer from the plate in kW.

$$T_{avg} = \frac{120 + 20}{2} = 70^\circ\text{C}$$

From A-22, p. 798

$$\rho = 1.028 \text{ kg/m}^3 \quad k = .02881 \frac{\text{W}}{\text{m}^\circ\text{C}}$$

$$\nu = 1.995 \times 10^{-5} \text{ m}^2/\text{sec}$$

$$Pr = 0.7177$$

$$Re_L = \frac{VL}{\nu} = \frac{6 \text{ m/sec} (8 \text{ m})}{1.995 \times 10^{-5} \text{ m}^2/\text{sec}} = 2.406 \times 10^5$$

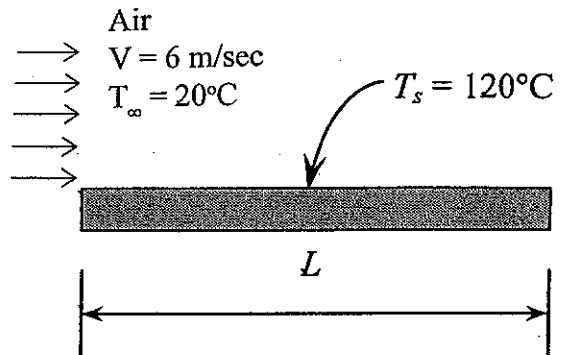
Turbulent at  $x = 8 \text{ m}$

$$Nu = (.037 Re_L^{.8} - 871) Pr^{.43} = 3,439.29 = \frac{hL}{k}$$

$$h = \frac{.02881}{8} (3439.29) = 12.386 \text{ W/m}^2 \cdot \text{K}$$

$$A_s = 2.5 \times 8 = 20 \text{ m}^2$$

$$\begin{aligned} \dot{Q} &= hA_s(T_s - T_\infty) = 12.386 \frac{\text{W}}{\text{m}^2 \cdot \text{K}} (20) \text{m}^2 (120 - 20) \\ &= 24,771.5 \text{ W} \quad \text{or} \quad \underline{24.77 \text{ kW}} \end{aligned}$$



2. Wind at  $17^\circ\text{C}$  is blowing directly across a 0.6 cm diameter wire of a transmission line at 40 km/hr. The rate of heat generated in the electrical line per meter length is 5 W/m. Neglect radiation effects and determine a) the convective heat transfer coefficient  $h$ , and b) the steady state surface temperature of the wire  $T_s$ . Use the temperature of  $20^\circ\text{C}$  to determine the properties of air.

AT  $20^\circ\text{C}$  From A-22, p. 798

$$\rho = 1.204 \text{ kg/m}^3$$

$$k = .02514 \text{ W/m}\cdot\text{C}$$

$$\nu = 1.516 \times 10^{-5} \text{ m}^2/\text{sec}$$

$$Pr = 0.7309$$

$$Re_D = \frac{VD}{\nu} = \frac{40(1000)(.006)}{1.516 \times 10^{-5}} = 4397.54$$

From Table 12-3, p. 555  $Nu = 0.193 Re^{.618} Pr^{1/3}$

$$h = \frac{k}{D} Nu = \frac{.02514}{.006} (31.03) = 129.985 \text{ W/m}^2\cdot\text{C}$$

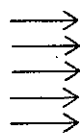
$$= 31.03$$

$$A = .006 \pi \text{ m}^2/\text{m}$$

$$\dot{W} = 5 \frac{\text{W}}{\text{m}} = 129.985 (.006) \pi (T_s - 17)$$

$$T_s = 19.04^\circ\text{C}$$

Wind  
 $V = 40 \text{ km/h}$   
 $T_\infty = 17^\circ\text{C}$



Transmission wire,  $T_s$   
 $D = 0.6 \text{ cm}$

3 Electronic components are housed in a box 32 cm high, 32 cm wide and 2 m long. Air blows over the box perpendicular to the length at 20 °C and 2 m/sec. The surface temperature of the box is not to exceed 60°C. Determine the total power rating of the electronic devices that can be housed in the box.

$$T_{avg} = \frac{60 + 20}{2} = 40^\circ\text{C}$$

$$k = 0.02662 \text{ W/m}\cdot^\circ\text{C}$$

$$\nu = 1.702 \times 10^{-5} \text{ m}^2/\text{sec}$$

$$Pr = 0.7255$$

$$Re_D = \frac{VD_h}{\nu} = \frac{(2)(.32)}{1.702 \times 10^{-5}} = 37,603$$

$$Nu = \frac{hD}{k} = 0.102 Re_D^{.675} Pr^{1/3} = 112.31 \quad (\text{Table 12-3})$$

p. 555

$$h = \frac{k}{D} Nu = 9.336 \text{ W/m}^2\cdot^\circ\text{C}$$

$$A = 4(.32)2$$

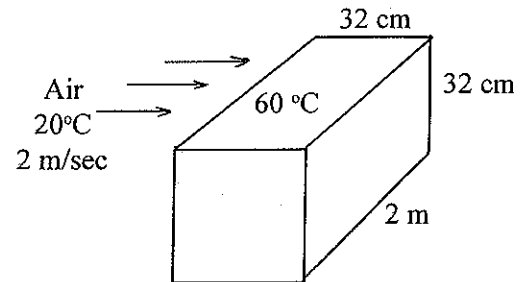
$$= 2.56 \text{ m}^2$$

$$\dot{Q} = hA_s(T_s - T_\infty) = 9.336(2.56)(40)$$

$$= \underline{955.98 \text{ W}}$$

If the ends are included  $A = 2.56 + 2(.32)^2 = 2.765 \text{ m}^2$

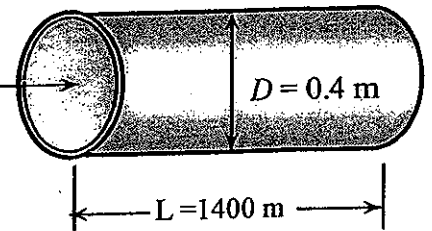
$$\dot{Q} = 9.336(2.765)(40) = \underline{1032.5 \text{ W}}$$



4. Engine oil initially at  $20^\circ\text{C}$  flows through a 40-cm-diameter pipe at an average velocity of 0.6 m/sec. A 1400-m-long section of the pipe is exposed to cold water at  $0^\circ\text{C}$ . Assume the pipe surface is also at  $0^\circ\text{C}$ , and disregarding the thermal resistance of the metal pipe walls, determine a) the mass flow rate through the pipe, b) the temperature of the oil,  $T_e$  as it leaves the 1400-m section, c) the log mean temperature difference,  $\Delta T_{lm}$  and d) rate of heat transfer from the oil. Properties of the engine oil can be evaluated at  $20^\circ\text{C}$ .

From Table A-19, p 794

$V = 0.6 \text{ m/sec}$   
 $T = 20^\circ\text{C}$



$$\rho = 888.1 \text{ kg/m}^3 \quad Pr = 10,863$$

$$\nu = 9.429 \times 10^{-4} \text{ m}^2/\text{sec} \quad k = 0.145 \frac{\text{W}}{\text{m}\cdot^\circ\text{C}}$$

$$C_p = 1881 \text{ J/kg}\cdot^\circ\text{C}$$

$$Re_D = \frac{VD}{\nu} = \frac{0.6(.4)}{9.429 \times 10^{-4}} = 254.53 \quad \text{laminar}$$

$$a) \quad \dot{m} = \rho A_c V_{avg} = 888.1 (\pi) \frac{.4^2}{4} (.6) = \underline{66.96 \text{ kg/sec}}$$

$$L_t = .05 Re_D Pr D = .05 (254.53)(10,863)(.4) = 55,299$$

Boundary layer has not developed

from equation 13-62

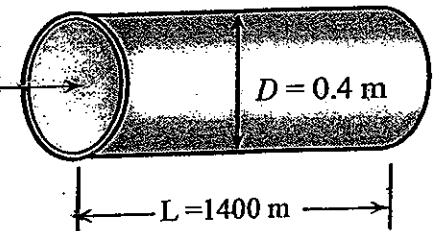
$$Nu = \frac{hD}{k} = 3.66 + \frac{.065 (D/L) Re_D Pr}{1 + .04 [(D/L) Re_D Pr]^{2/3}} = 15.286$$

$$h = \frac{k}{D} Nu = \frac{0.145}{.4} (15.286) = 5.541 \text{ W/m}\cdot^\circ\text{C}$$

4. Engine oil initially at  $20^{\circ}\text{C}$  flows through a 40-cm-diameter pipe at an average velocity of  $0.6\text{ m/sec}$ . A 1400-m-long section of the pipe is exposed to cold water at  $0^{\circ}\text{C}$ . Assume the pipe surface is also at  $0^{\circ}\text{C}$ , and disregarding the thermal resistance of the metal pipe walls, determine a) the mass flow rate through the pipe, b) the temperature of the oil,  $T_e$  as it leaves the 1400-m section, c) the log mean temperature difference,  $\Delta T_{\ln}$  and d) rate of heat transfer from the oil. Properties of the engine oil can be evaluated at  $20^{\circ}\text{C}$ .

$$A_s = \pi DL = \pi(.4)(1400) \\ = 1759.29 \text{ m}^2$$

$$V = 0.6 \text{ m/sec} \\ T = 20^{\circ}\text{C}$$



$$\dot{m} = 66.96 \text{ kg/sec} \quad C_p = 1881 \text{ J/kg}^{\circ}\text{C}$$

$$h = 5.541 \text{ W/m}^2\text{C}$$

$$b) \quad T_e = T_s - (T_s - T_i) e^{-\frac{hA}{\dot{m}C_p}} \\ = 0 - (0 - 20) e^{-\frac{(5.54)(1759)}{1881(66.96)}} = \underline{18.51^{\circ}\text{C}}$$

$$c) \quad \Delta T_{\ln} = \frac{T_e - T_i}{\ln\left[\frac{T_s - T_e}{T_s - T_i}\right]} = \underline{19.24^{\circ}\text{C}}$$

$$d) \quad \dot{Q}_{\text{avg}} = hA_s \Delta T_{\ln} = 5.541(1759.29)(19.24) \\ = 187,546 \text{ W} \\ = 187.55 \text{ kW}$$

5. A 40-W resistance heater 60 cm long is placed along the center of a cylinder that has a diameter  $D = 2$  cm. The ends are insulated and the surface emissivity  $\epsilon = 0.1$ . The average surface temperature of the cylinder is maintained at  $100^\circ\text{C}$  while the room temperature is at a constant  $20^\circ\text{C}$ . Assume that radiation from the environment is also at  $20^\circ\text{C}$ . For steady state operation, determine a) the Rayleigh number, b) the Nusselt number from Table 14-1, c) the amount of heat transferred by free convection, and d) the amount transferred by radiation.

$$T_{\text{avg}} = \frac{100 + 20}{2} = 60^\circ\text{C}$$

From Table A-22

$$k = .0281 \frac{\text{W}}{\text{m}^\circ\text{C}} \quad Pr = .7202$$

$$\nu = 1.896 \times 10^{-5} \text{ m}^2/\text{sec} \quad \beta = \frac{1}{T} = .003 \text{ K}^{-1} \quad \delta = .02 \text{ m}$$

$$Ra = \frac{g \beta (T_s - T_{\text{surr}}) \delta^3 Pr}{\nu^2} = 37,735$$

From Table 14.1, Eq 14.25

$$Nu = \left\{ .6 + \frac{.387 Ra^{1/6}}{\left[ 1 + \left( \frac{.559}{Pr} \right)^{9/16} \right]^{4/27}} \right\}^2 = 6.064 = \frac{h \delta}{k}$$

$$e) h = \frac{6.064}{.02} (.0281) = 8.514 \text{ W/m}^2\text{ }^\circ\text{C} \quad A = \pi DL = .0377 \text{ m}^2$$

$$\dot{Q}_{\text{conv}} = h A \Delta T = 25.68 \text{ W}$$

$$b) \dot{Q}_{\text{rad}} = \epsilon \pi D L \sigma (T_s^4 - T_{\text{surr}}^4) \\ = 0.1 \pi (.02) .6 (5.67 \times 10^{-8}) (373^4 - 293^4) \\ = 2.56 \text{ W}$$

